

# Energetical background of common approaches in geomechanics

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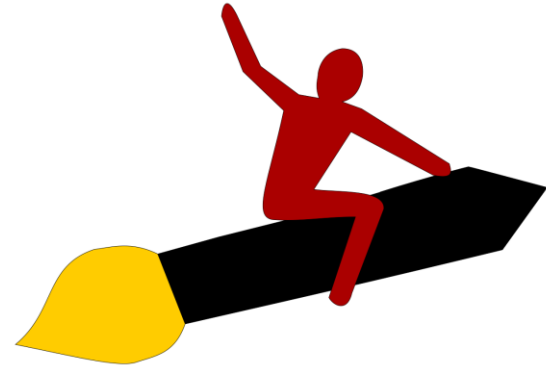
# Outlook

- ▶ **Derivation of balance equations and stress measures**
- ▶ **Localized deformation**
- ▶ **Elastoplasticity**
- ▶ **Anisotropy**
- ▶ **Coupling**



# Derivation of the balance equations

- ▶ *The laws of physics are invariant under a transformation between two coordinate frames moving at a constant velocity with respect to each other.*

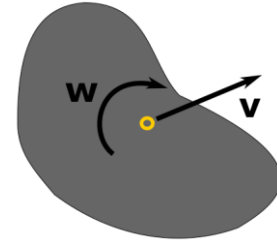


# Conserved quantities

$$\dot{\tilde{E}} = \dot{W}_F + \dot{W}_C - \dot{E}_{kin}$$

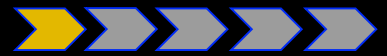
- ▶  $W_F$  is the work of the forces
- ▶  $W_C$  is the work of the couples
- ▶  $E_{kin}$  is the kinetic energy

For a single, rigid object this means:



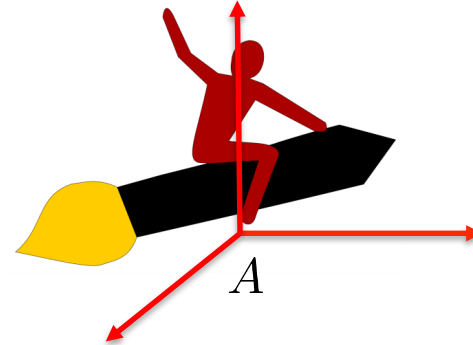
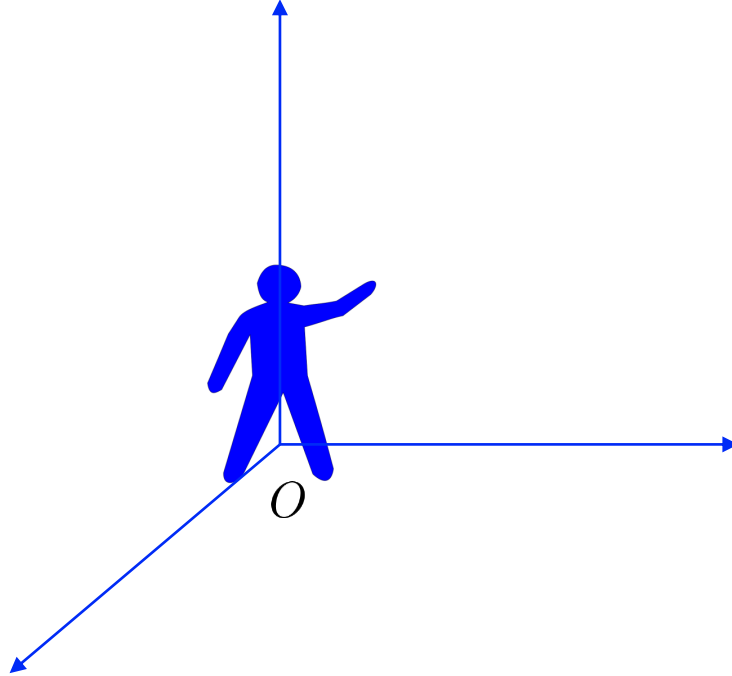
$$\dot{W}_F + \dot{W}_C = \mathbf{f} \cdot \mathbf{v} + \mathbf{m} \cdot \mathbf{w}$$

$$\dot{E}_{kin} = \frac{1}{2} \dot{m} \mathbf{v} \cdot \mathbf{v} + m \mathbf{v} \cdot \dot{\mathbf{v}} + \frac{1}{2} \mathbf{w}^T \underline{\theta} \mathbf{w} + \mathbf{w}^T \underline{\theta} \dot{\mathbf{w}}$$

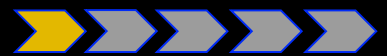


# Galilean change of observer

- ▶ A second observer moves at constant linear velocity with respect to the initial system



$$\vec{OA} = at$$



## Galilean change of observer

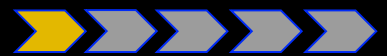
- ▶ A second observer moves at constant linear velocity with respect to the initial system

$$\underline{\theta}' = \underline{\theta} \quad \& \quad \underline{w}' = \underline{w} \quad \& \quad \underline{m}' = \underline{m} \quad \& \quad \underline{v}' = \underline{v} - \underline{a}$$

- ▶ Then it must hold that:

$$\begin{aligned} \left( \dot{W}_F + \dot{W}_C - \dot{E}_{kin} \right)_0 &= \left( \dot{W}_F + \dot{W}_C - \dot{E}_{kin} \right)_a \\ \underline{f} \cdot \underline{v} + \underline{m} \cdot \underline{w} - \underline{f}' \cdot \underline{v}' - \underline{m} \cdot \underline{w} &= \frac{1}{2} \dot{m} \underline{v} \cdot \underline{v} + m \underline{v} \cdot \dot{\underline{v}} + \\ + \frac{1}{2} \underline{w}^T \dot{\underline{\theta}} \underline{w} + \underline{w}^T \underline{\theta} \dot{\underline{w}} - \frac{1}{2} \dot{m} \underline{v}' \cdot \underline{v}' - m \underline{v}' \cdot \dot{\underline{v}}' - \frac{1}{2} \underline{w}^T \dot{\underline{\theta}} \underline{w} - \underline{w}^T \underline{\theta} \dot{\underline{w}} &\Rightarrow \end{aligned}$$

$$\underline{f}' \cdot \underline{a} = (\underline{f}' - \underline{f}) \cdot \underline{v} + \dot{m} \underline{a} \cdot \underline{v} + m \underline{a} \cdot \dot{\underline{v}} - \frac{1}{2} \dot{m} \underline{a} \cdot \underline{a}$$



## Galilean change of observer

- ▶ Since  $\mathbf{a}$  is arbitrarily selected,

$$\mathbf{f}' \cdot \mathbf{a} = (\mathbf{f}' - \mathbf{f}) \cdot \mathbf{v} + \dot{m}\mathbf{a} \cdot \mathbf{v} + m\mathbf{a} \cdot \dot{\mathbf{v}} - \frac{1}{2}\dot{m}\mathbf{a} \cdot \mathbf{a}$$

- ▶ means that the following holds:

- ▶ Force transformation rule

$$\mathbf{f}' = \mathbf{f}$$

- ▶ Mass balance

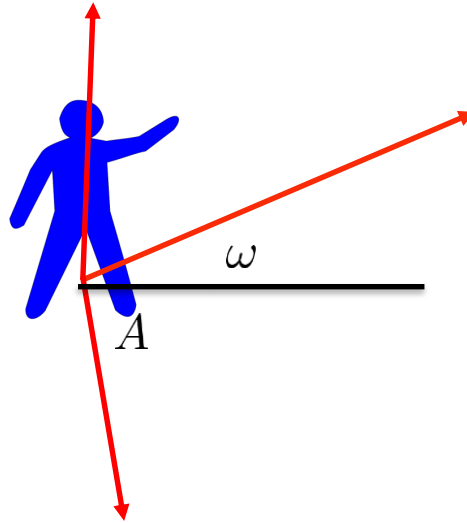
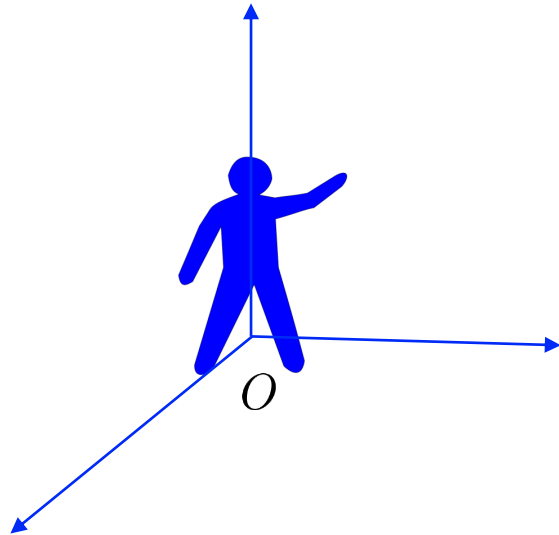
$$\dot{m} = 0$$

- ▶ Momentum balance

$$\mathbf{f} = \dot{m}\mathbf{v} + m\dot{\mathbf{v}}$$

# Leibniz change of observer

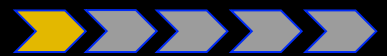
- ▶ A second observer moves at constant angular velocity with respect to the initial system



$$\omega = \dot{\theta} t$$

$$\left( \dot{W}_F + \dot{W}_C - \dot{E}_{kin} \right)_0 = \left( \dot{W}_F + \dot{W}_C - \dot{E}_{kin} \right)_o$$



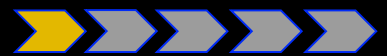


## Leibniz change of observer

- ▶ *A second observer moves at constant angular velocity with respect to the initial system*

$$\begin{aligned}
 \dot{E}'_{kin} &= \frac{1}{2} \dot{m} \left[ \underline{\dot{\mathbf{O}}}^T \mathbf{x} + \underline{\mathbf{O}}^T \mathbf{v} \right]^T \left[ \underline{\dot{\mathbf{O}}}^T \mathbf{x} + \underline{\mathbf{O}}^T \mathbf{v} \right] + \\
 &+ m \left[ \underline{\dot{\mathbf{O}}}^T \mathbf{x} + \underline{\mathbf{O}}^T \mathbf{v} \right]^T \left[ \underline{\dot{\mathbf{O}}}^T \underline{\mathbf{W}}^T \mathbf{x} + 2 \underline{\dot{\mathbf{O}}}^T \mathbf{v} + \underline{\mathbf{O}}^T \dot{\mathbf{v}} \right] + \\
 &+ \frac{1}{2} \left[ \mathbf{w}^T - \mathbf{b}^T \right] \underline{\mathbf{O}} \underline{\dot{\theta}'} \underline{\mathbf{O}}^T \left[ \mathbf{w} - \mathbf{b} \right] + \\
 &+ \frac{1}{2} \left[ \mathbf{w}^T - \mathbf{b}^T \right] \underline{\mathbf{O}} \underline{\dot{\theta}'} \left[ \underline{\dot{\mathbf{O}}}^T (\mathbf{w} - \mathbf{b}) + \underline{\mathbf{O}}^T \dot{\mathbf{w}} \right] + \\
 &+ \frac{1}{2} \left[ (\mathbf{w}^T - \mathbf{b}^T) \underline{\dot{\mathbf{O}}} + \dot{\mathbf{w}}^T \underline{\mathbf{O}} \right] \underline{\dot{\theta}'} \underline{\mathbf{O}}^T \left[ \mathbf{w} - \mathbf{b} \right]
 \end{aligned}$$

where  $\underline{\mathbf{O}}$  is a rotation tensor.



## Leibniz change of observer

- ▶ The previous equation yields the already known mass balance and
- ▶ The couple transformation rule

$$\mathbf{m}' = \mathbf{m}$$

- ▶ the angular inertia tensor balance

$$\underline{\dot{\theta}} = \underline{\mathbf{W}}\underline{\theta} + \underline{\theta}\underline{\mathbf{W}}$$

- ▶ and the angular momentum balance

$$\mathbf{m} = \underline{\dot{\theta}}\mathbf{w} + \underline{\theta}\dot{\mathbf{w}}$$

- ▶ where  $\mathbf{W}$  is the rotational velocity tensor corresponding to the vector  $\mathbf{w}$ .

# Micromechanical stress tensors – two particles in contact

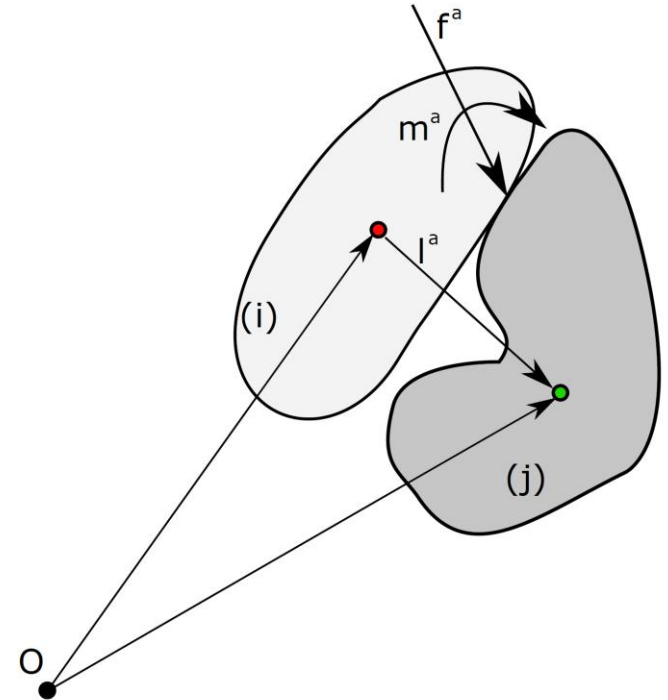
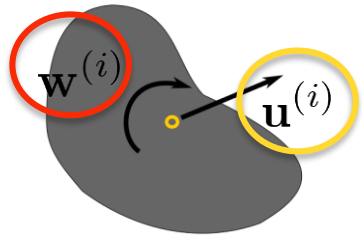
▶ Consider the depicted grains and their contact.

▶ The relative displacement at the contact is:

$$\mathbf{u}^{(i,j)} = \mathbf{u}^{(i,a)} - \mathbf{u}^{(j,a)}$$

▶ with

$$\mathbf{u}^{(i,a)} = \mathbf{u}^{(i)} + \mathbf{w}^{(i)} \times (\mathbf{x}^{(a)} - \mathbf{x}^{(i)})$$



# Micromechanical stress tensors

- ▶ *The same principle can be applied to assemblies of rigid particles, such as granular media*

## Assumption:

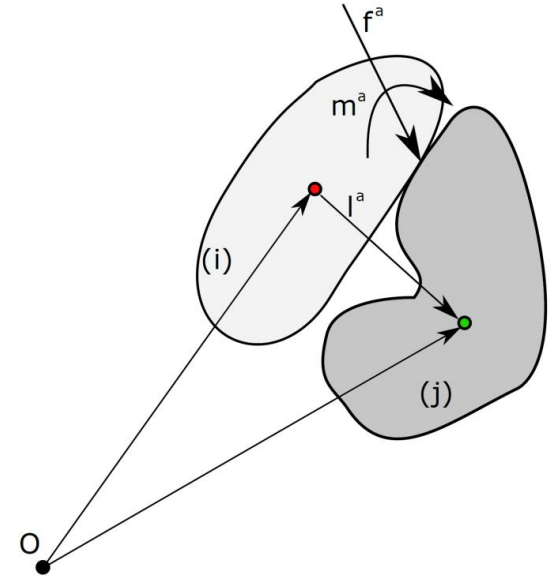
- ▶ The displacement and rotation rates are affine:

$$\mathbf{u}^{(i)} = \mathbf{u}^O + \nabla \mathbf{u}^O \cdot \mathbf{x}^{(i)}$$

$$\mathbf{w}^{(i)} = \mathbf{w}^O + \nabla \mathbf{w}^O \cdot \mathbf{x}^{(i)}$$

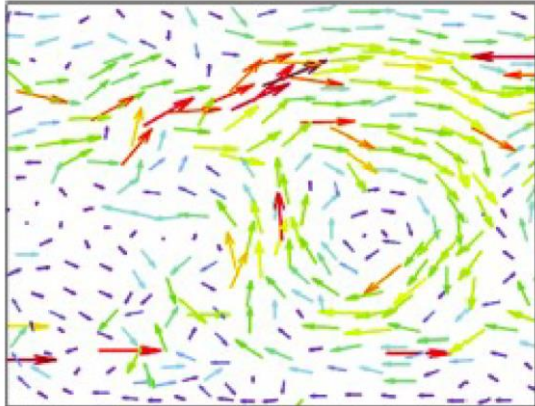
- ▶ meaning that

$$P_{int} = \sum_{a \in \mathcal{C}} \left( (\mathbf{f}^a \otimes \mathbf{l}^a) : (\nabla \mathbf{v}^O - \underline{\mathbf{W}}^O) \right) + \sum_{a \in \mathcal{C}} \left( (\mathbf{f}^a \times \mathbf{l}^a) \otimes (\mathbf{x}^a - \mathbf{x}^O) : \nabla \mathbf{w}^O \right) + \sum_{a \in \mathcal{C}} \left( \mathbf{m}^a \otimes \mathbf{l}^a : \nabla \mathbf{w}^O \right)$$



# Affinity of displacements and rotations

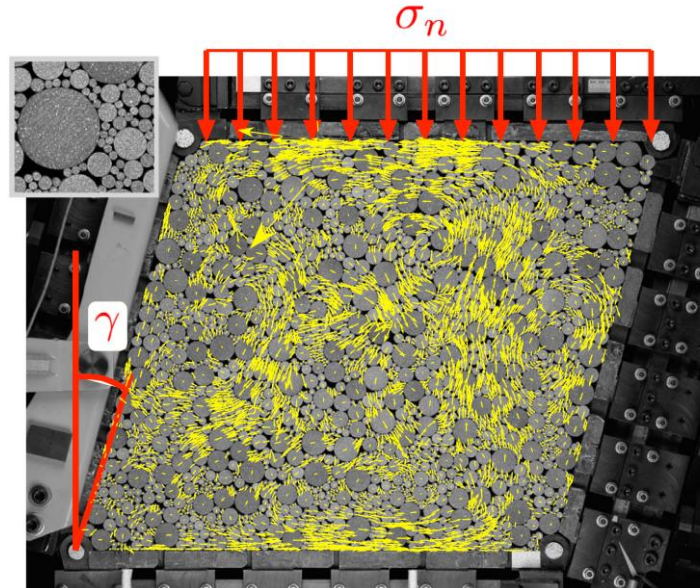
## *DEM Simulation Velocities*



*Miller et al. (2013)*

## *Experiment*

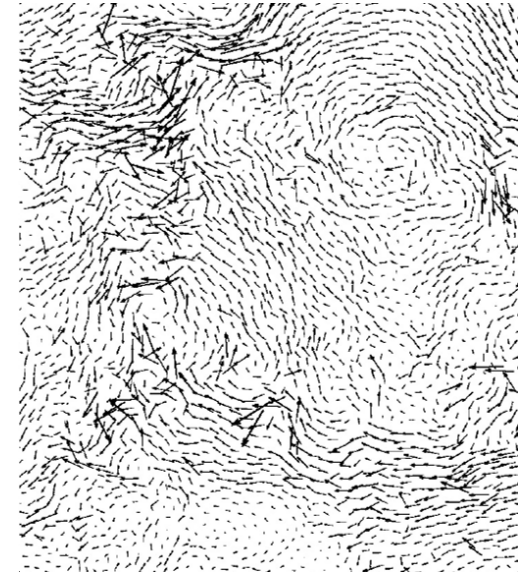
## *Displacement Fluctuations*



*Combe et al. (2015)*

## *DEM Simulation*

## *Velocity Fluctuations*



*Radjai and Roux (2002)*



# Micromechanical stress tensors

**Assumption:**  $\bar{P}_{int} = V \left( \underline{\sigma} : \underline{\dot{\Gamma}} + \underline{\mu} : \underline{\dot{\kappa}} \right)$

▶ meaning that

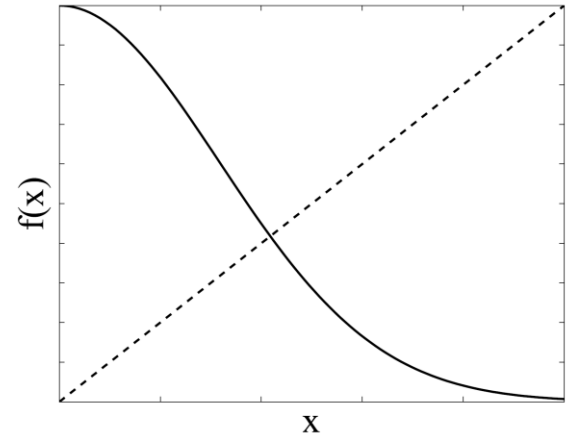
$$\underline{\sigma} = \frac{1}{V} \sum_{a \in \mathcal{C}} \mathbf{f}^a \otimes \mathbf{l}^a$$

$$\underline{\mu} = \frac{1}{V} \sum_{a \in \mathcal{C}} (\mathbf{m}^a \otimes \mathbf{l}^a) + \frac{1}{V} \sum_{a \in \mathcal{C}} ((\mathbf{f}^a \times \mathbf{l}^a) \otimes \underline{\mathbf{x}}^a)$$

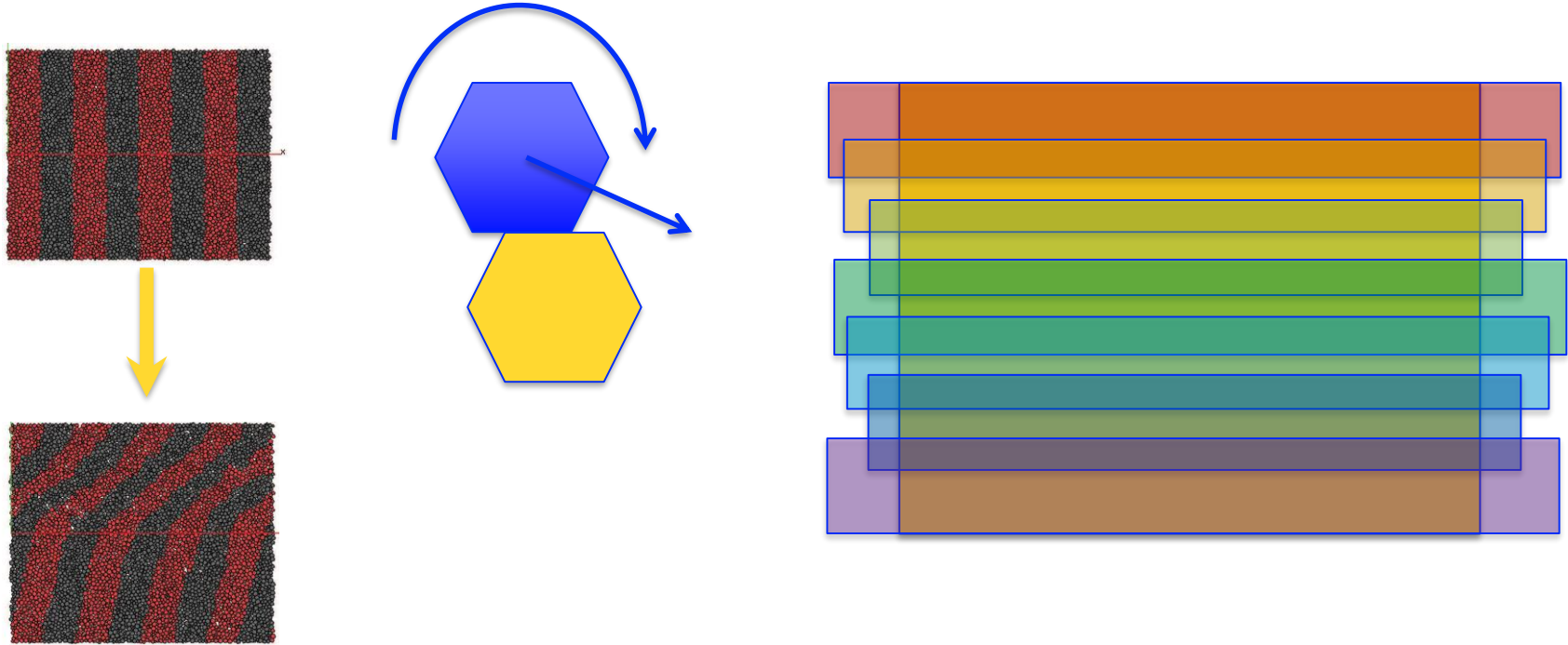
▶ alternatively

$$\underline{\mu}' = \frac{1}{V} \sum_{a \in \mathcal{C}} (\mathbf{m}^a \otimes \mathbf{l}^a) + \frac{1}{V} \sum_{a \in \mathcal{C}} ((\mathbf{f}^a \times \mathbf{l}^a) \otimes \mathbf{l}^a)$$

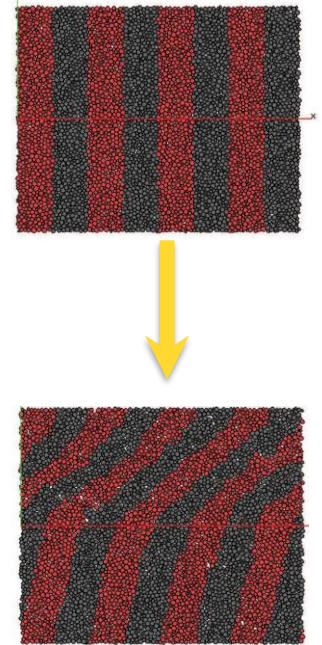
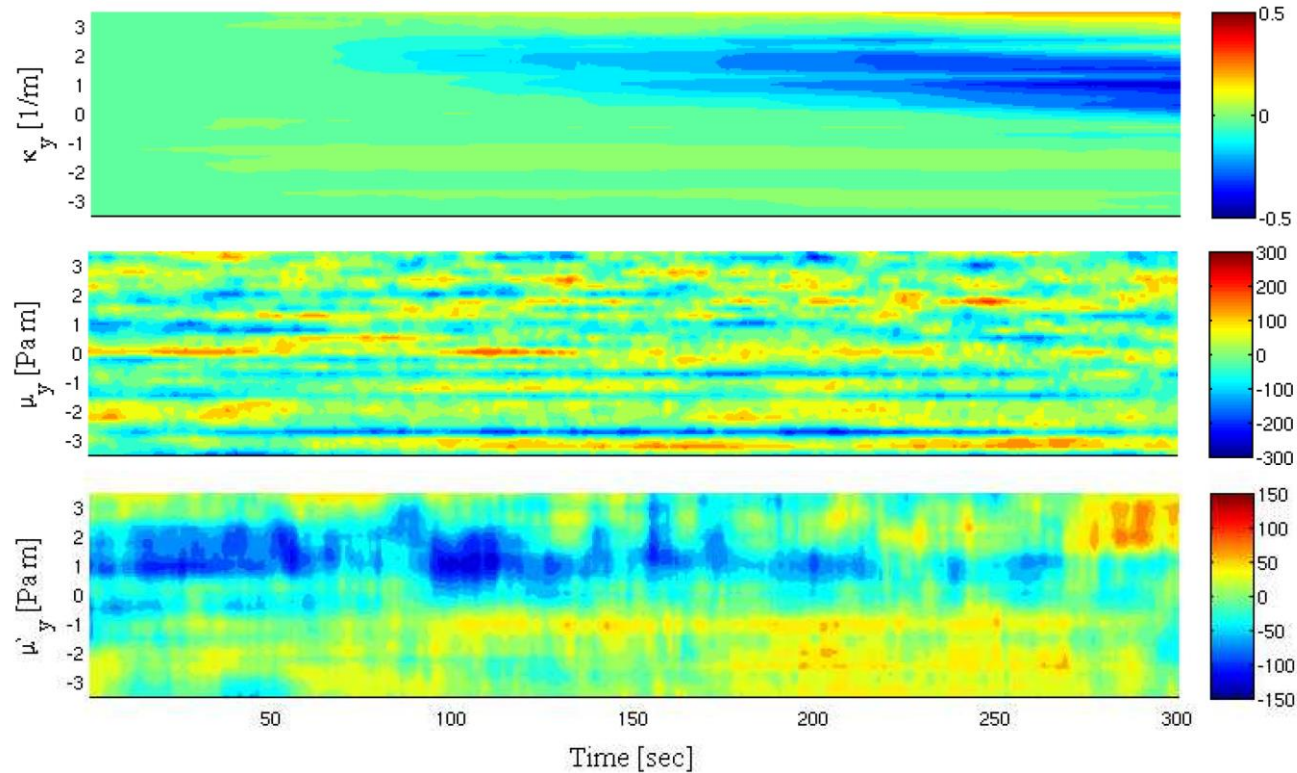
***Tordesillas and Walsh (2002)***



# Comparison of different formulations



# Comparison of different formulations

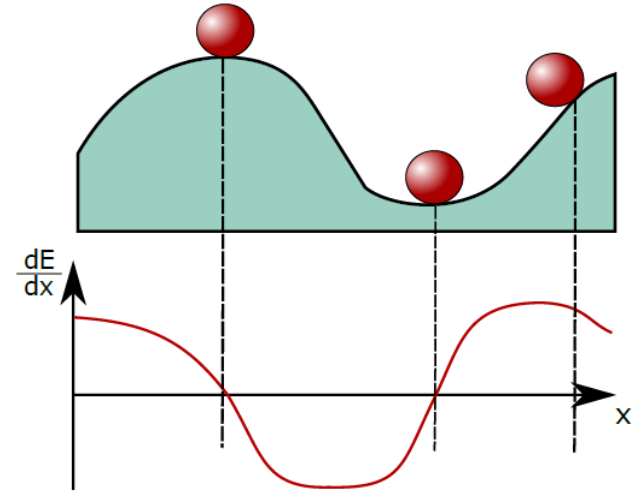






## Stability – potential energy characteristics

- ▶ A force field exists
- ▶ To move something in the force field, work must be done
- ▶ The force field is conservative
- ▶ The force field itself does negative work when another force is moving something against it
- ▶ It is recoverable energy



**Minimum potential energy -> Stable equilibrium**

# Stability – Equivalence of virtual work and balance equation

- ▶ The solution coincides with the one of the virtual work method
- ▶ The balance equations read

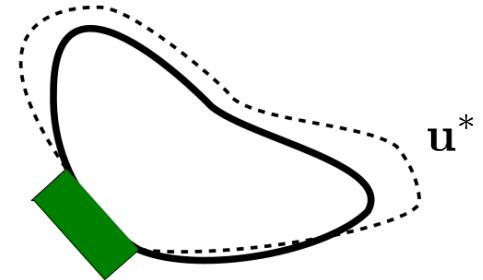
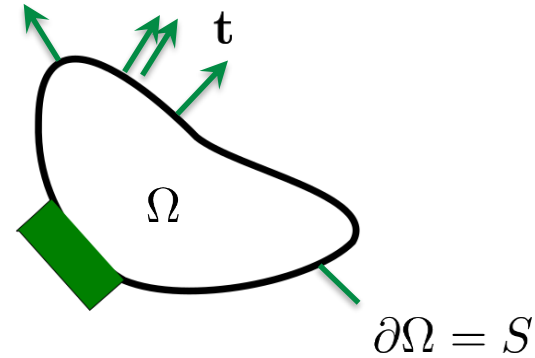
$$\sigma_{ij,j} + f_i = 0 \Leftrightarrow (\sigma_{ij,j} + f_i) u_i^* = 0 \Leftrightarrow$$

$$(\sigma_{ij} u_i^*)_{,j} - \sigma_{ij} u_{i,j}^* + f_i u_i^* = 0 \Leftrightarrow$$

$$\sigma_{ij} \epsilon_{ij}^* = (\sigma_{ij} u_i^*)_{,j} + f_i u_i^*$$

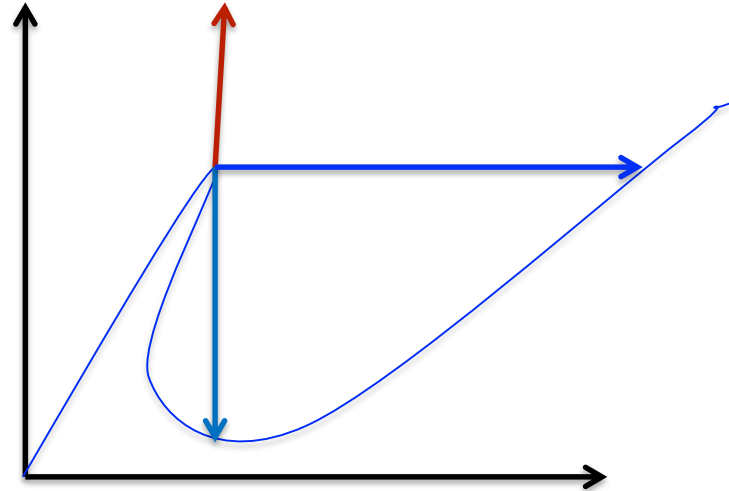
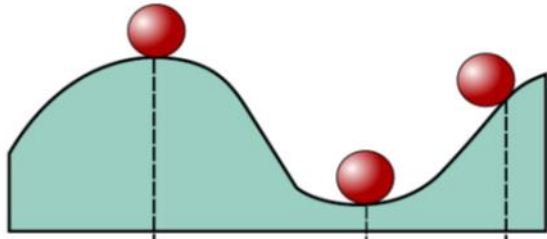
- ▶ Integrating over the domain and using the divergence theorem

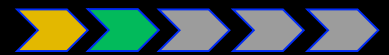
$$\int_{\Omega} \sigma_{ij} \epsilon_{ij}^* d\omega = \int_{\omega} f_i u_i^* d\omega + \int_S t_i u_i^* ds$$



# Stability – Mathematically?

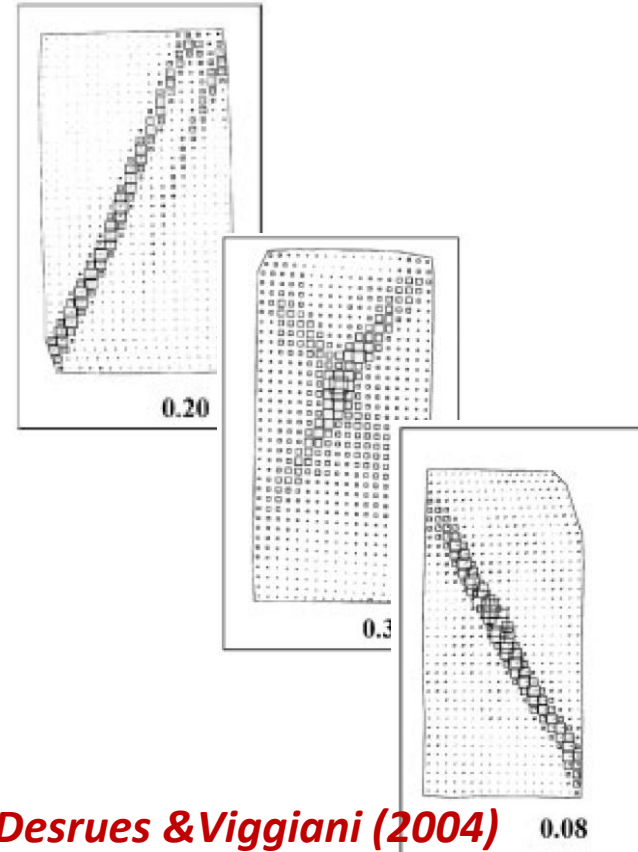
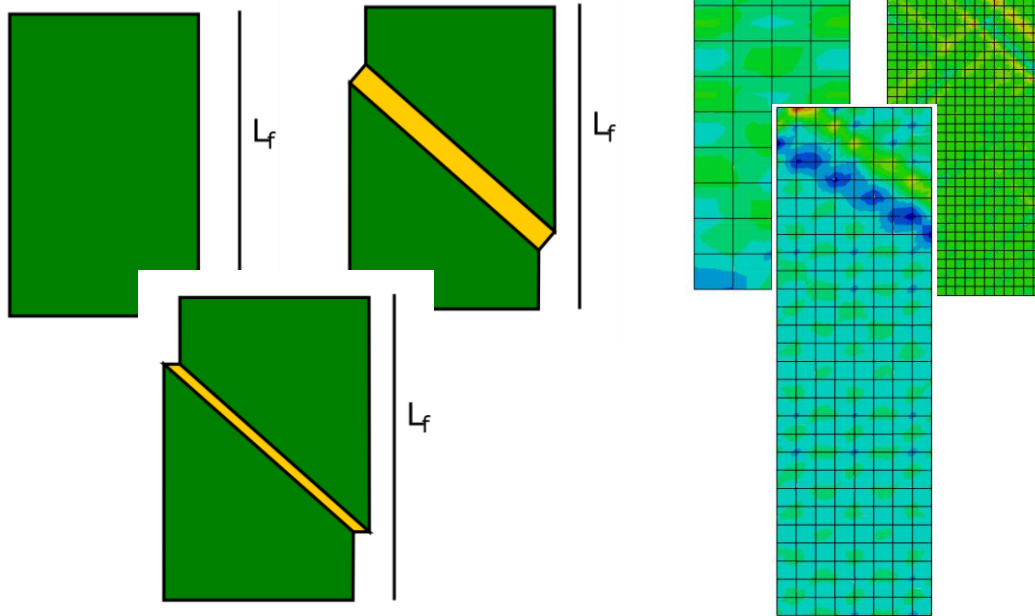
- ▶ A problem is well posed when
  - ▶ There is a solution
  - ▶ The solution is unique
  - ▶ The solution's behavior changes continuously with the initial conditions
- ▶ Deviations lead to numerical instability
- ▶ This has nothing to do with energy





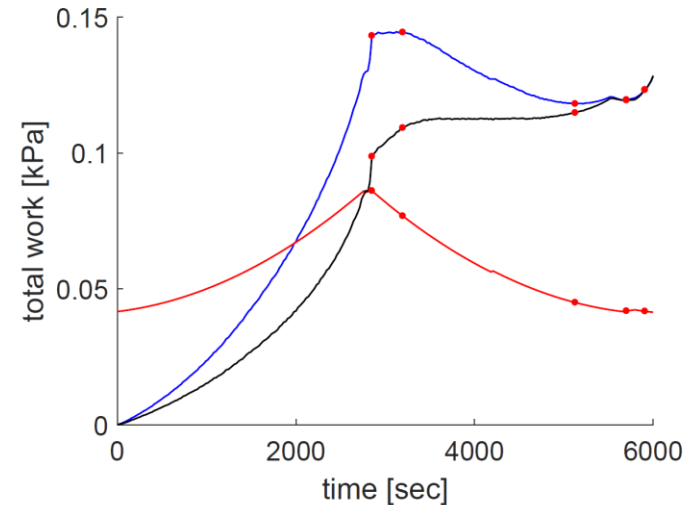
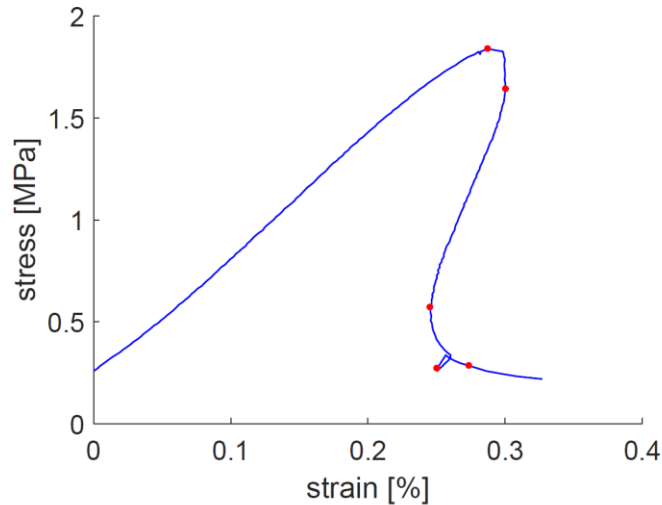
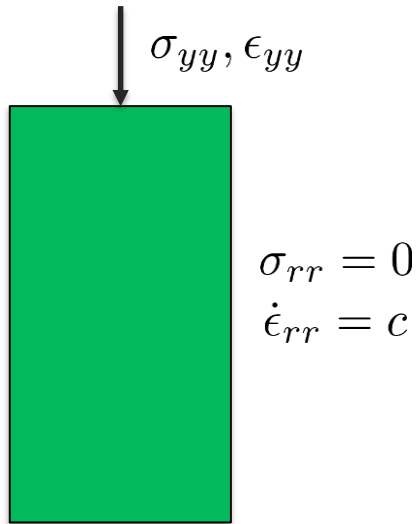
# Strain localization

- ▶ More than one possible solutions



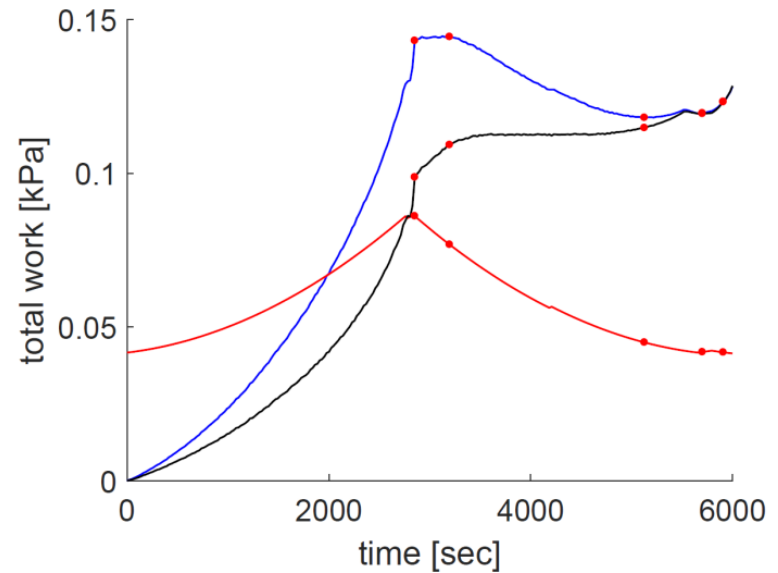
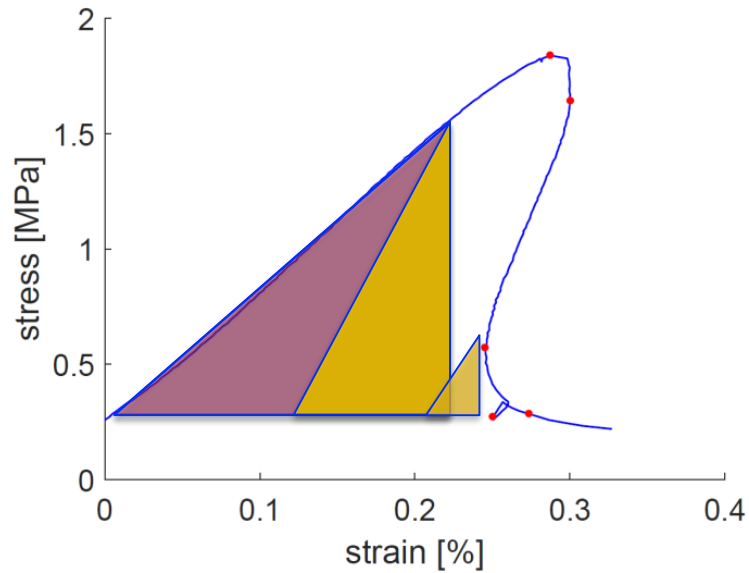
# And the energy?

- ▶ A simple example: uniaxial test with radial strain rate control



# And the energy?

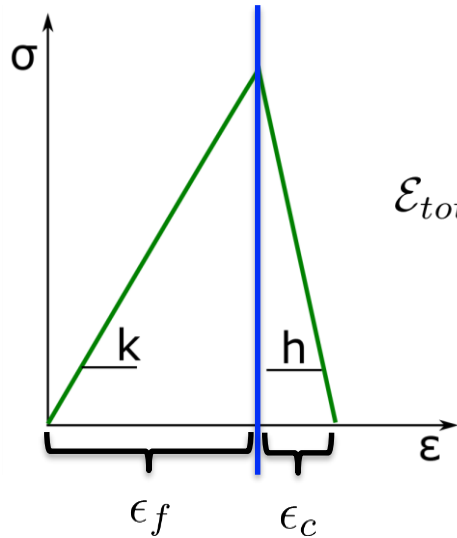
- ▶ A simple example: uniaxial test with radial strain rate control



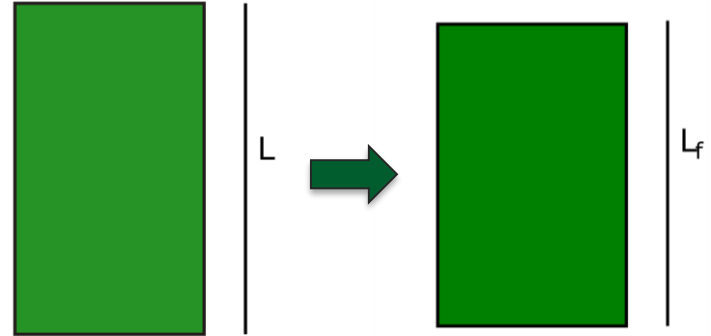


# Strain localization – Thought example in 1D

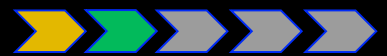
- ▶ Consider the constitutive response of the material point to be the one shown here.



$$\mathcal{E}_{tot} = \frac{1}{2}\sigma_f\epsilon_f + \frac{1}{2}\sigma_f\epsilon_c$$

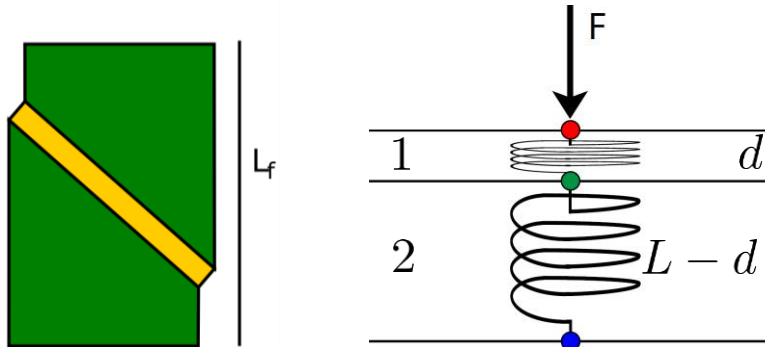


$$\epsilon_f + \epsilon_c = \frac{L - L_f}{L}$$



## Strain localization – Thought experiment in 1D

- ▶ Assume a shear band forms with a width of  $d$ . Two springs in series can be viewed as the mechanical equivalent.



$$\sigma = \sigma_1 + \sigma_2$$

$$\epsilon = \frac{\epsilon_1 d + \epsilon_2 (L - d)}{L}$$

- ▶ The mechanical response for each spring is

$$\sigma_1 = k\epsilon_1 - (k + h) < \epsilon_1 - \epsilon_f >$$

$$\sigma_2 = k\epsilon_2$$

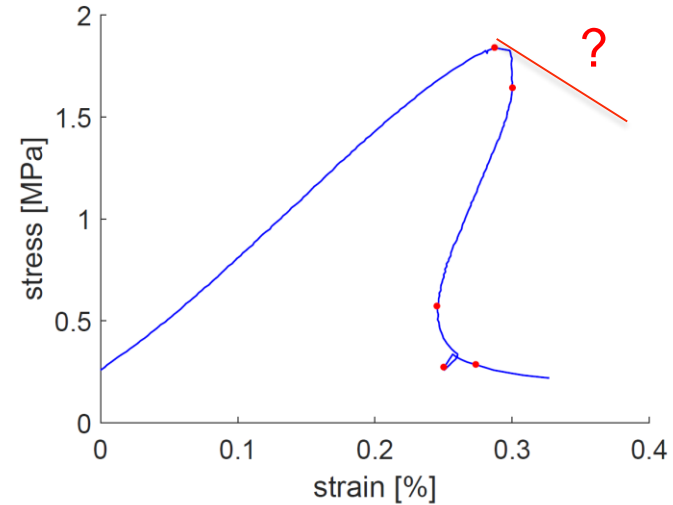
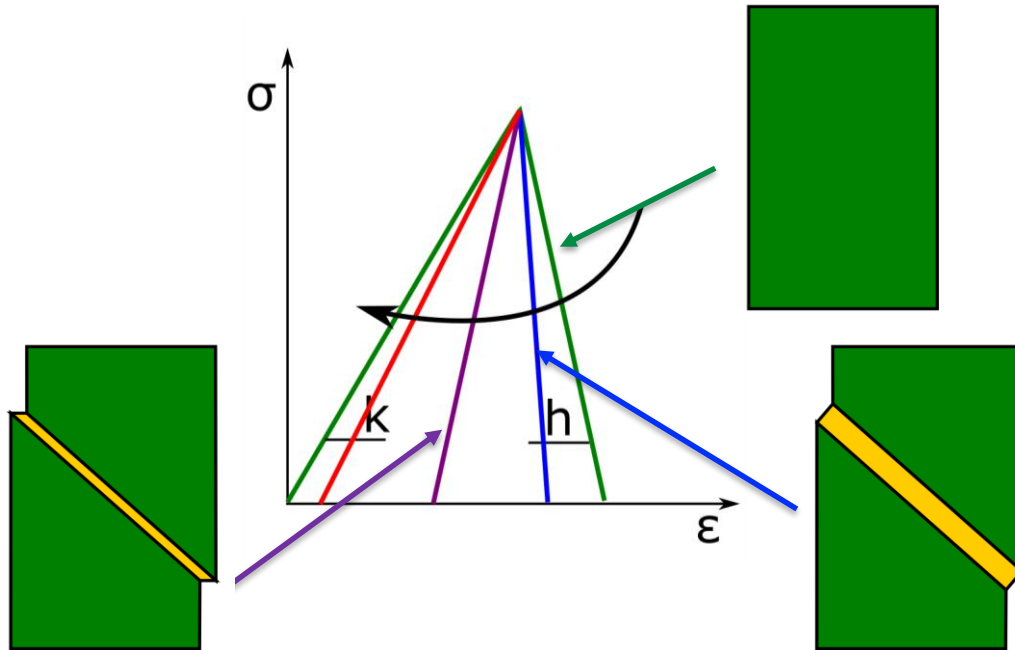
- ▶ The stress becomes zero when  $\epsilon_1 = \epsilon_f + \epsilon_c$
- ▶ Meaning that it becomes zero at

$$\epsilon = (\epsilon_f + \epsilon_c) \frac{d}{L}$$



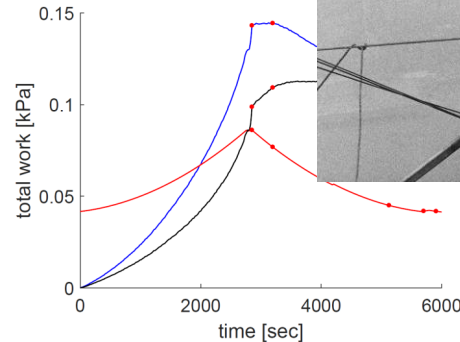
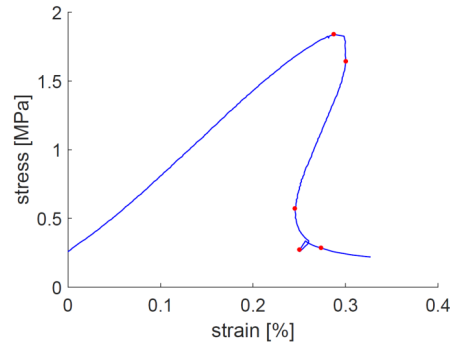
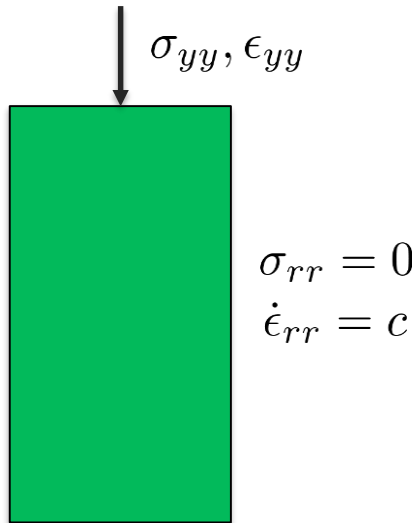
# Strain localization – Thought experiment in 1D

- ▶ For different (decreasing) values of  $d$ :



# Controllability is another thing:

- ▶ A simple example: uniaxial test with radial strain rate control





# Elasto-plasticity

- ▶ It is assumed that deformations are reversible (elastic) within a limited domain

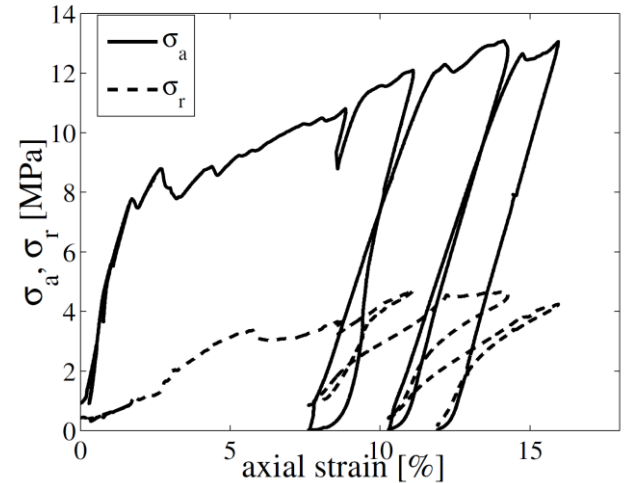
$$f(\underline{\sigma}) < 0$$

- ▶ Strain rates are decomposed into reversible and irreversible:

$$\dot{\epsilon}_{ij} = \dot{\epsilon}_{ij}^e + \dot{\epsilon}_{ij}^p$$

- ▶ To solve the problem, the direction of the plastic strain increment is required. It is assumed that

$$\dot{\epsilon}_{ij}^{pl} = \dot{\lambda} \frac{\partial g}{\partial \sigma_{ij}}$$

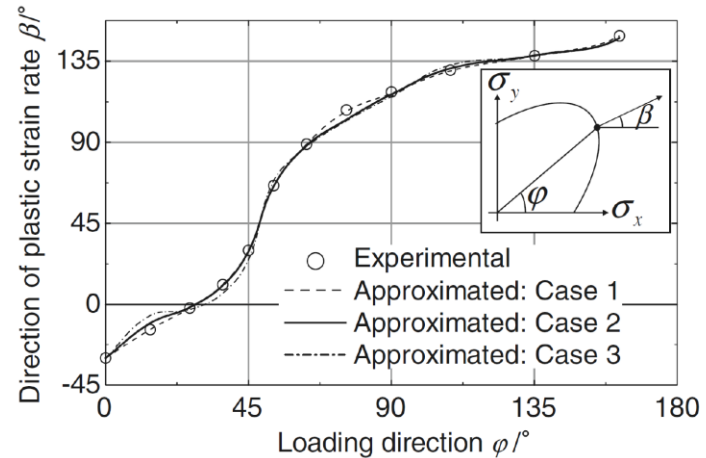
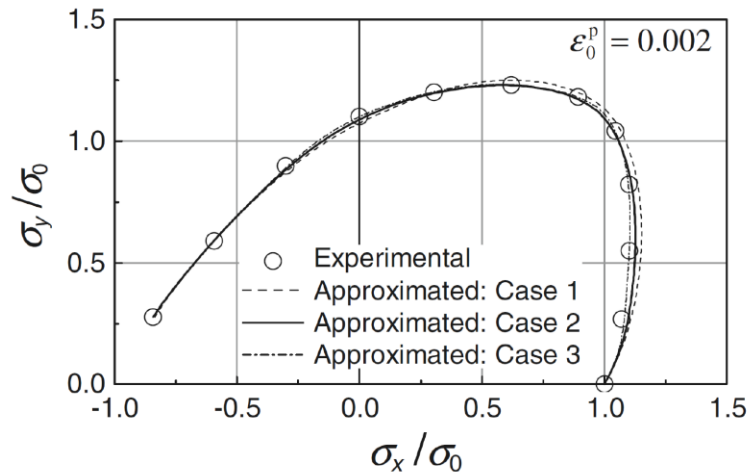


**Castellanza et al. (2009)**



# Associativity

▶ The flow rule reads:  $\dot{\epsilon}_{ij}^{pl} = \dot{\lambda} \frac{\partial f}{\partial \sigma_{ij}}$



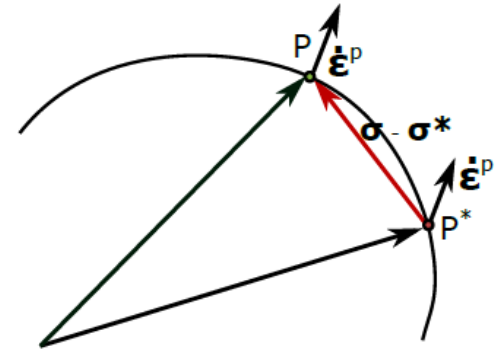
***Ishiki et al. (2011)***

## Convexity & associativity

- ▶ **Assumption:** The stress state is such, that the dissipation rate is maximum (Hill 1948)
- ▶ For normality, the dissipation rate is maximum with respect to the stress, if the yield surface is convex.

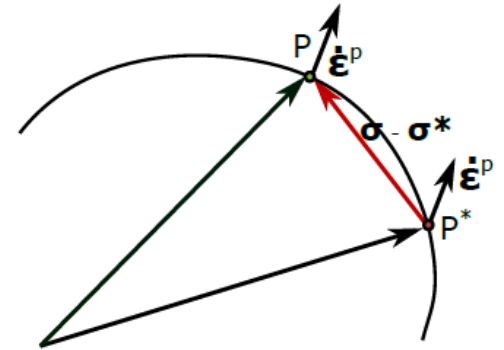
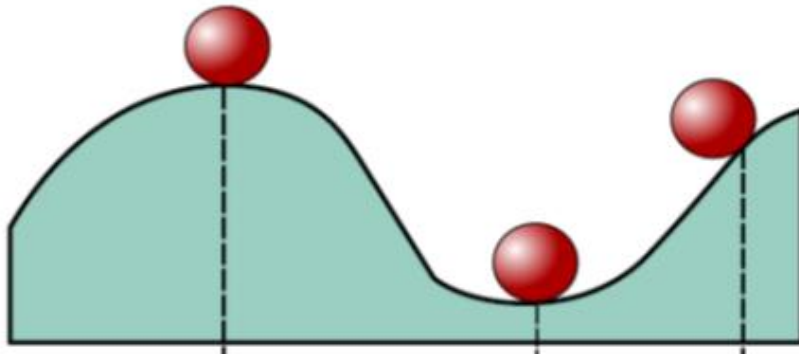
$$\dot{W}_p = (\sigma_{ij} - \sigma_{ij}^*) \dot{\epsilon}_{ij}^p \geq 0$$

- ▶ which would mean that
  - ▶ the flow rule is associative
  - ▶ the yield surface is convex
- ▶ The underlying assumption is that the body tries to minimize its internal energy as fast as possible.

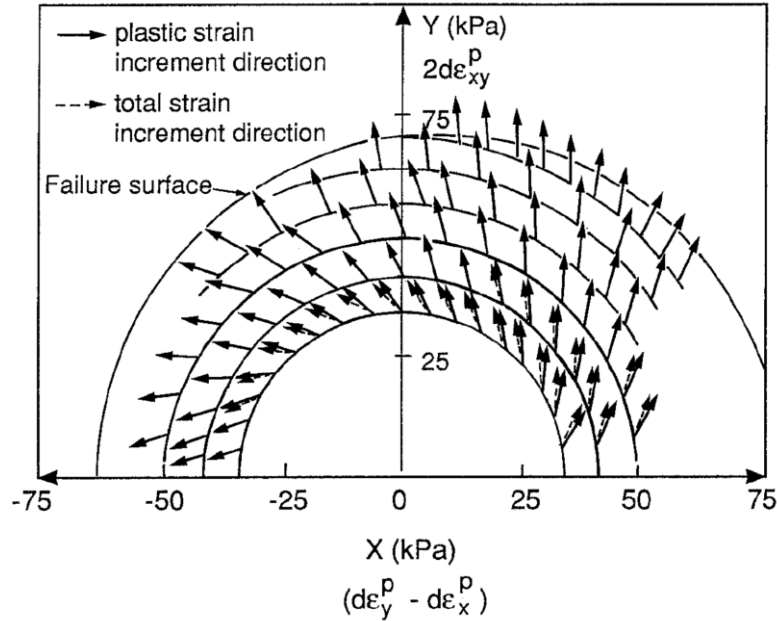


# Convexity & associativity

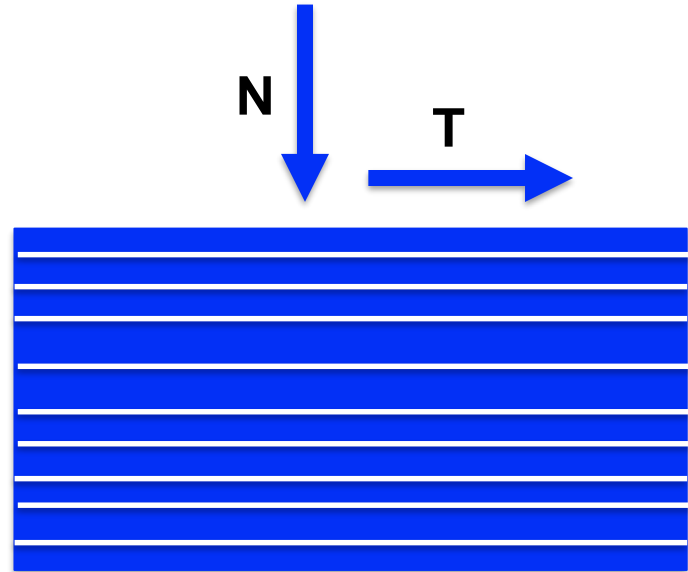
- ▶ **Assumption:** The body tries to minimize its internal energy as fast as possible.



# Non associativity



***Gutierrez and Ishihara (2000)***

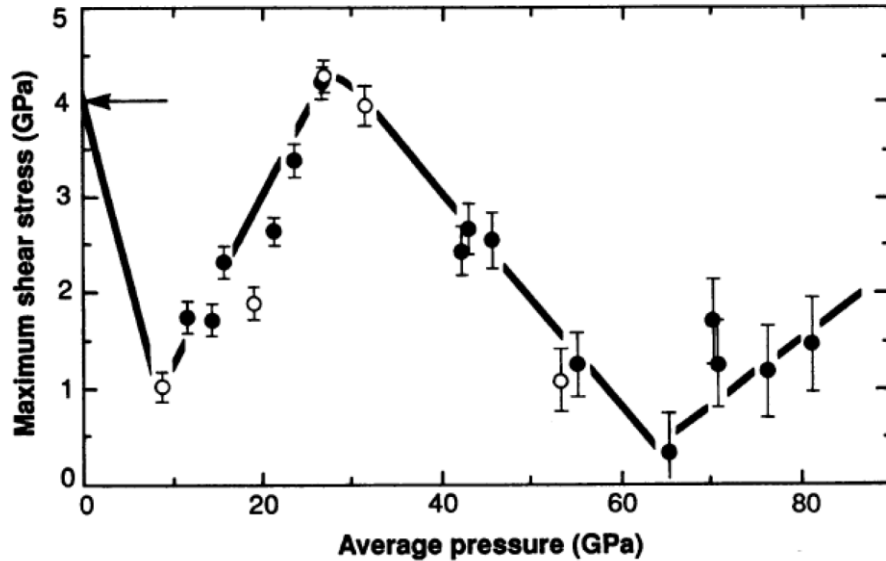


No volumetric deformation



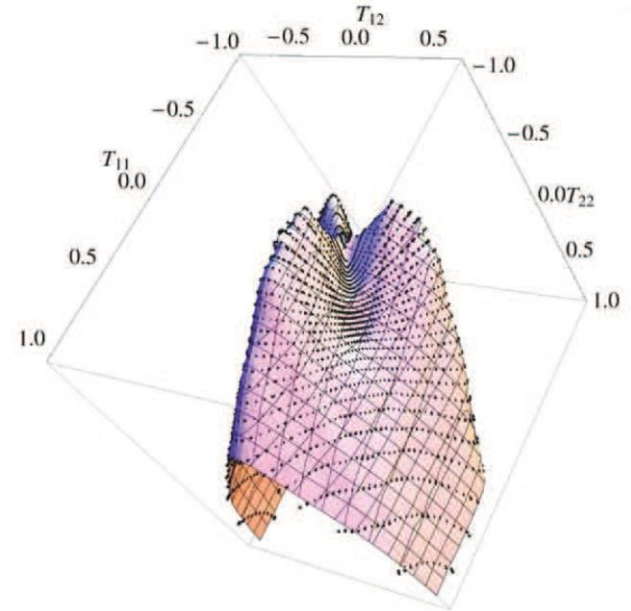
# Non convexity?

## Tests on fused silica glass



*Meade and Jeanloz (1988)*

## FEM on honeycombs



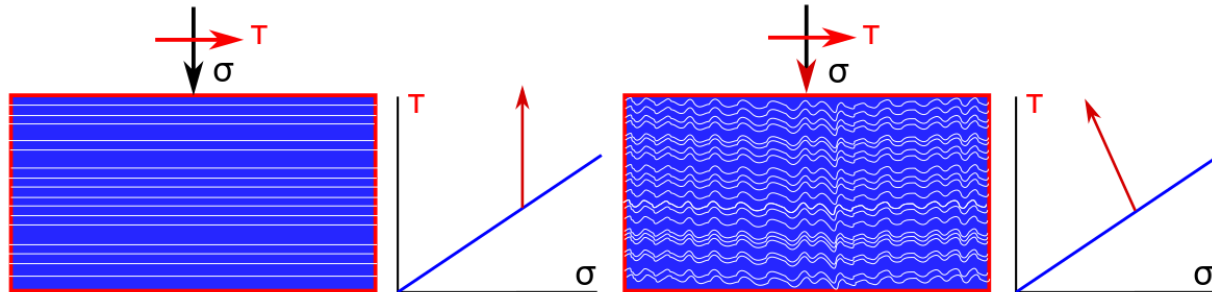
*Glüge and Bucci (2017)*





# Limitations

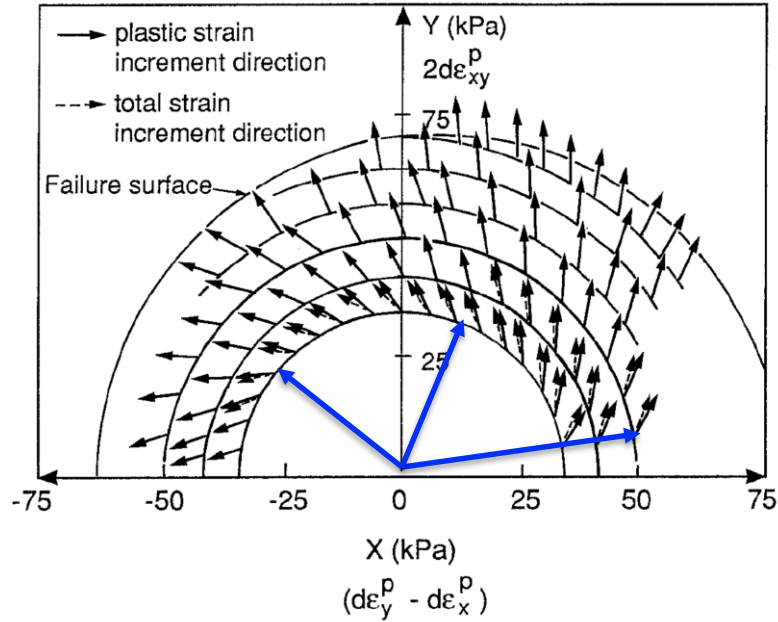
- ▶ The dissipation rate should always be non negative, or
- ▶ Dissipated energy along a closed loading path should be non-negative
- ▶ The angle between stress vector and plastic strain increment vector can never be more than  $90^\circ$



- ▶ Different types of constraints can determine the flow direction

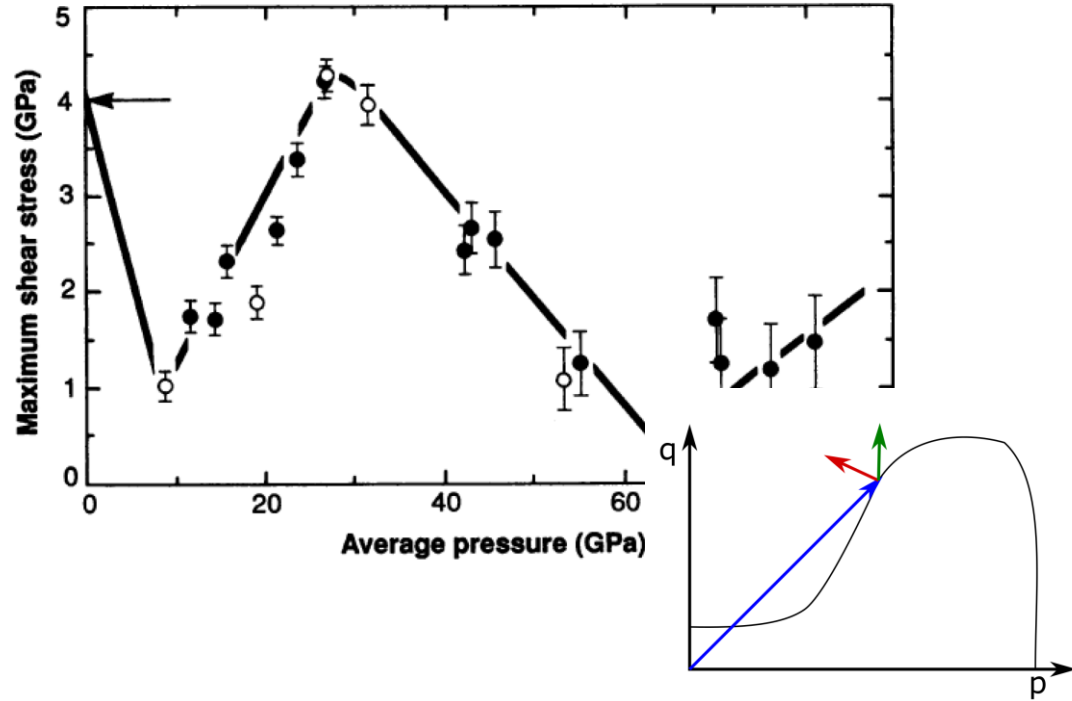


# Limitations



***Gutierrez and Ishihara (2000)***

## ***Meade and Jeanloz (1988)***





# Anisotropy

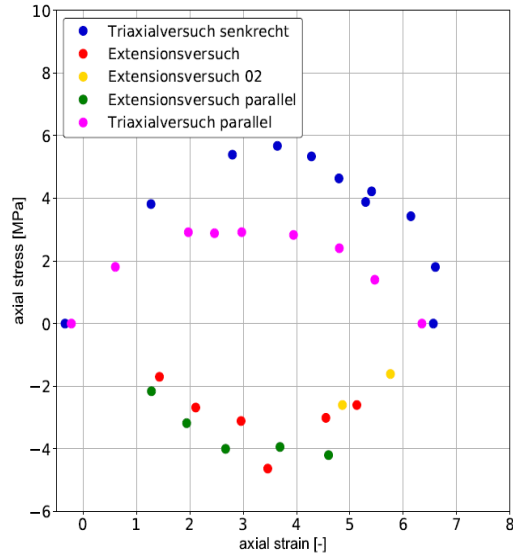
- ▶ Usually ignored because:
  - ▶ Usually not known
  - ▶ Experimentally hard/expensive to get
  - ▶ Already incorporated in the failure envelope from experimental data
  
- ▶ We will take a look at what this means for the elastic energy



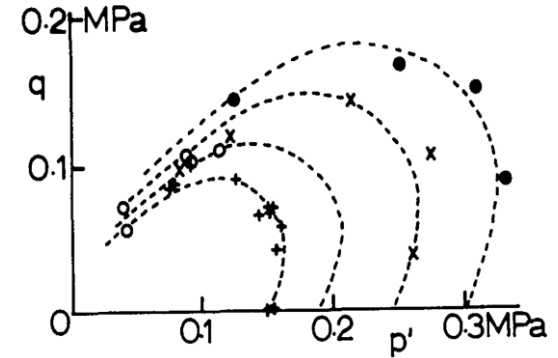


# Anisotropy – Experimental observations

► For cohesive materials the yield locus is affected:



*Courtesy of J. Leuthold*

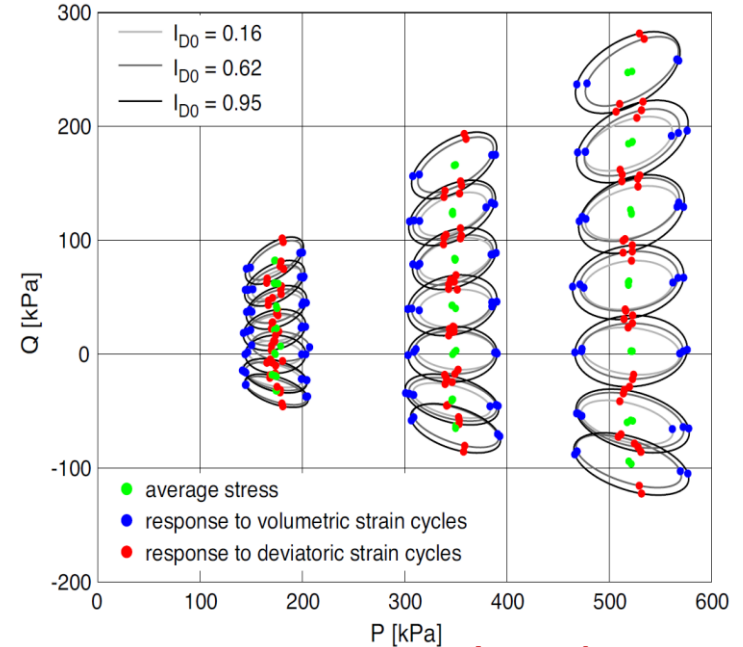
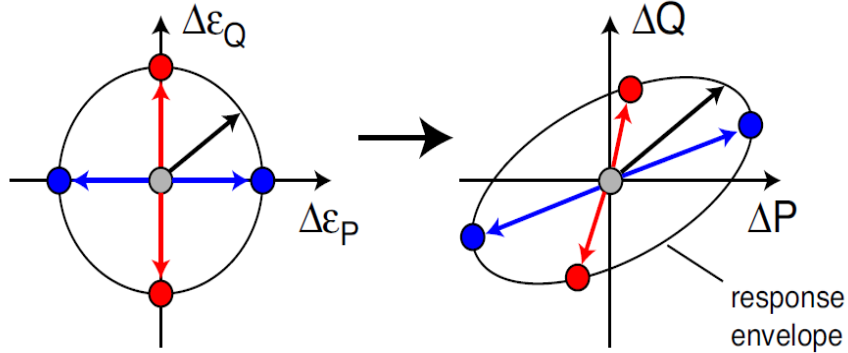


*Muir Wood and Graham (1990)*



# Anisotropy – Experimental observations

► For granular materials the elastic response is affected:



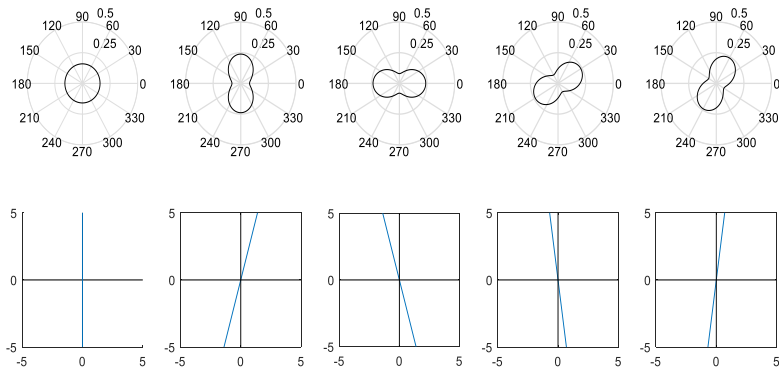
**Wichtmann (2016)**



# Anisotropy

- ▶ To simplify matters, a 2-D elastic anisotropy is considered:

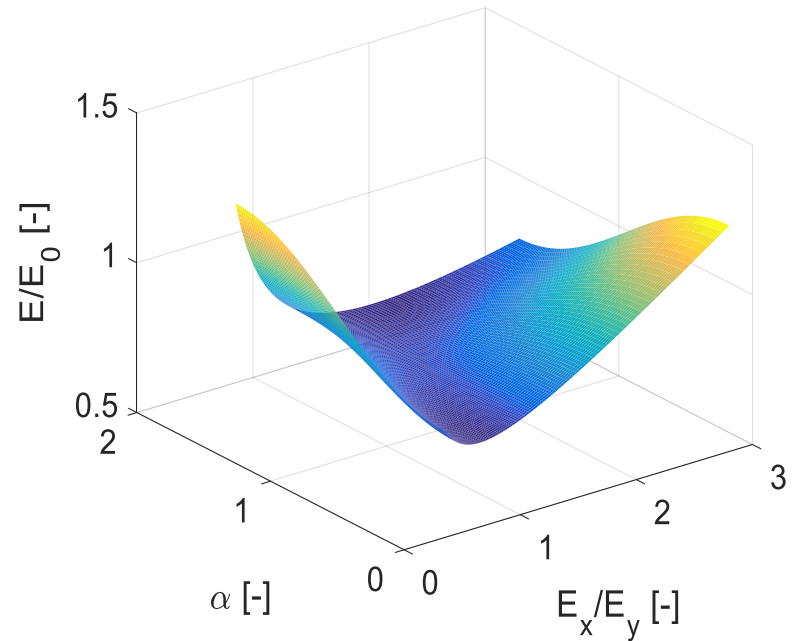
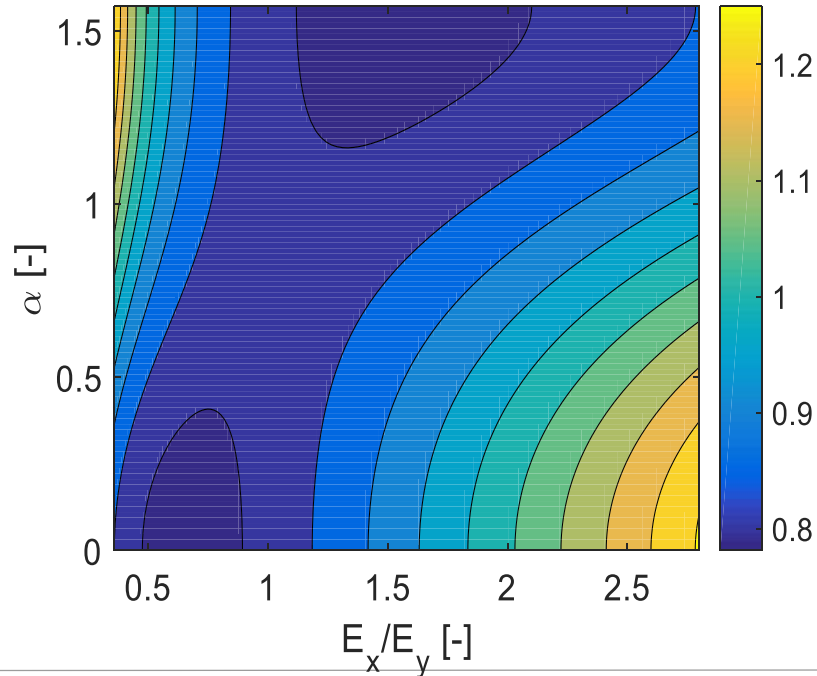
$$\begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{xy} \end{bmatrix} = \begin{bmatrix} \frac{1}{E_x} & -\frac{\nu_{yx}}{E_y} & 0 \\ -\frac{\nu_{xy}}{E_x} & \frac{1}{E_y} & 0 \\ 0 & 0 & \frac{1}{G_{xy}} \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{xy} \end{bmatrix}$$





# Anisotropy

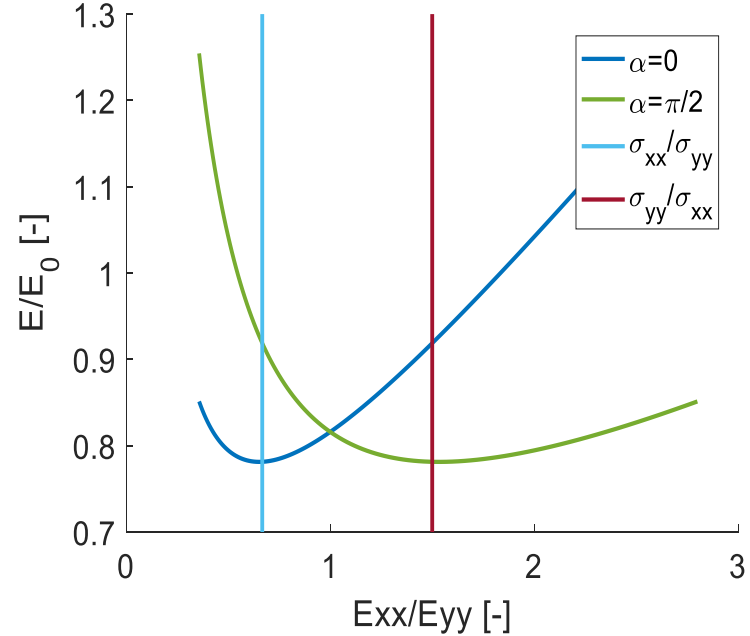
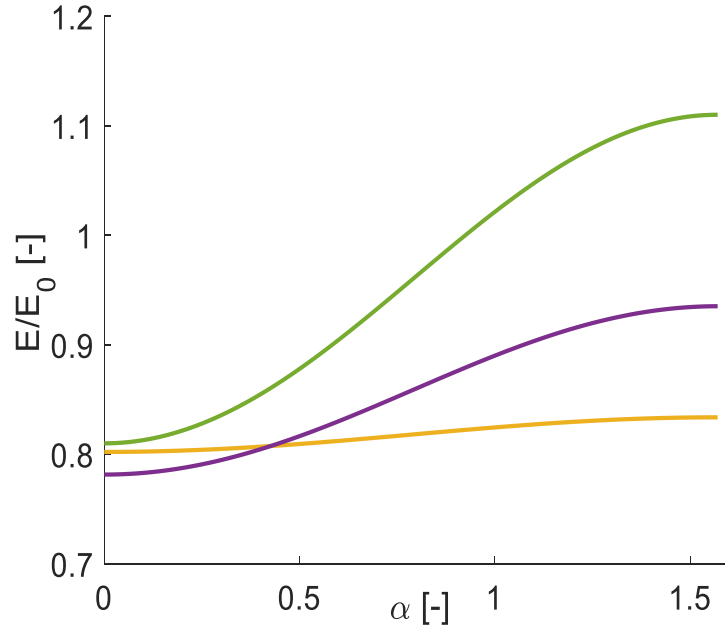
- ▶ The elastic energy depends on the relative angle and the degree of anisotropy:





# Anisotropy

- ▶ The minimum is found for coaxial tensors, when the stiffness ratio equals the stress ratio:



- ▶ In general granular media try to move in this direction





# Implications for modelling

- ▶ Elastic strains are miscalculated
  - ▶ Elastic strain increments are usually much smaller than the plastic strain increments
- ▶ The elastic energy is overestimated
  - ▶ It is usually of no direct consequence to the results or application
- ▶ Energy ‘invested’ in changing the internal structure is neglected
  - ▶ This may affect coaxiality, but does not play a role for monotonic coaxial loading



## Structure evolution: an example

- ▶ Work rate balance in general

$$\sigma_{ij}\dot{\epsilon}_{ij} = \dot{E}^{el} + D$$

- ▶ Ignoring the evolution of anisotropy

$$\begin{aligned}\sigma_{ij}\dot{\epsilon}_{ij} &= \sigma_{ij}\dot{\epsilon}_{ij}^{el} + D \Rightarrow \\ D &= \sigma_{ij}\dot{\epsilon}_{ij}^{pl}\end{aligned}$$

- ▶ Considering the evolution of anisotropy

$$\begin{aligned}\sigma_{ij}\dot{\epsilon}_{ij} &= \sigma_{ij}\dot{\epsilon}_{ij}^{el} + \frac{\partial E}{\partial \alpha}\dot{\alpha} + D \Rightarrow \\ D &= \sigma_{ij}\dot{\epsilon}_{ij} - \sigma_{ij}\dot{\epsilon}_{ij}^{el} - \frac{\partial E}{\partial \alpha}\dot{\alpha}\end{aligned}$$



## Coupling: Thermoelasticity

- ▶ The heat equation reads

$$\rho c \frac{\partial T}{\partial t} - \nabla \cdot (k \nabla T) = \dot{q}_v$$

- ▶ where  $\rho$  is the density
- ▶  $c$  is the specific heat capacity
- ▶  $T$  is the temperature
- ▶  $k$  is the thermal conductivity
- ▶  $\dot{q}_v$  is the volumetric heat source
- ▶ Temperature increase causes thermal expansion

$$\epsilon_T = -\alpha T$$

- ▶ where  $\alpha$  is the thermal expansion coefficient and compression is assumed positive

$$\rho c \frac{\partial T}{\partial t} = k \nabla^2 T$$

- ▶ for constant conductivity and no volumetric source



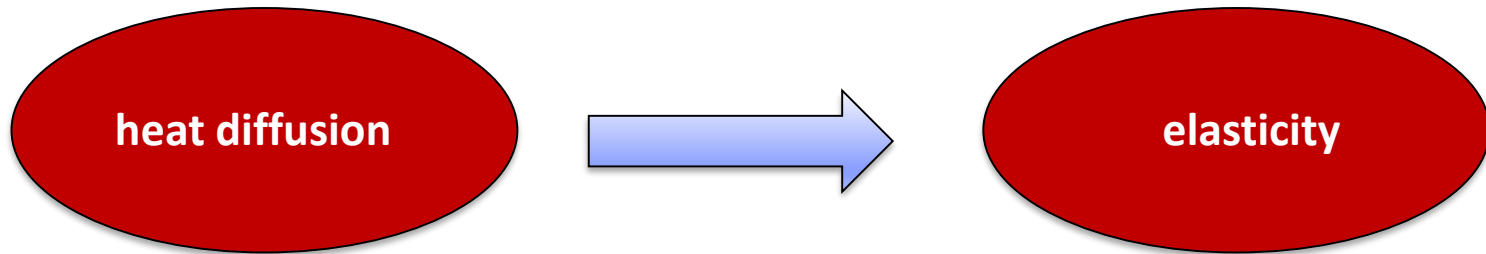
## Thermoelasticity – one way coupling

- ▶ The heat diffusion is assumed uncoupled from the elastic response:

$$\rho c \frac{\partial T}{\partial t} - \nabla \cdot (k \nabla T) = \dot{q}_v$$

- ▶ The elastic response depends on the (independently evaluated) temperature change

$$\boldsymbol{\sigma} = \underline{\mathbf{E}} (\boldsymbol{\epsilon} - \alpha T \mathbf{I})$$





## Thermoelasticity – a simple example

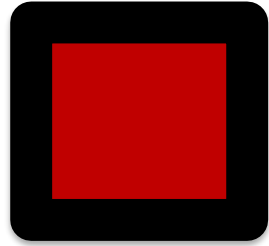
- ▶ Consider a small uniform volume.
- ▶ No boundary displacements are allowed.
- ▶ The temperature is increased from  $T_0$  to  $T_1$ .
- ▶ The heat equation becomes

$$\rho c \frac{\partial T}{\partial t} - \nabla \cdot (k \nabla T) = \dot{q}_v \Rightarrow \rho c \frac{\partial T}{\partial t} = \dot{q}_v \Rightarrow q_v = \rho c (T_1 - T_0)$$

- ▶ meaning that the energy density stored due to the temperature change is

$$Q = \rho c (T_1 - T_0)$$

- ▶ generated by the volumetric heat source





## Thermoelasticity – a simple example

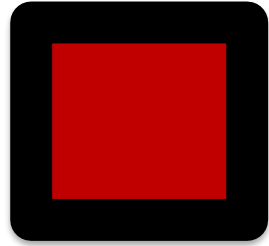
- ▶ The thermal expansion – since the material is constrained – causes an increase in mean pressure:

$$\boldsymbol{\sigma} = -\underline{\mathbf{E}}(\alpha(T_1 - T_0)\mathbf{I})$$

- ▶ increasing the elastic energy stored to

$$E = \frac{1}{2}\boldsymbol{\sigma} \cdot \boldsymbol{\epsilon} = \frac{1}{2}K\alpha^2(T_1 - T_0)^2$$

- ▶ Where did this come from?



$$Q = \rho c(T_1 - T_0)$$



# Thermoelasticity – coupled

- ▶ The heat equation is derived from the energy balance and Fourier's law:

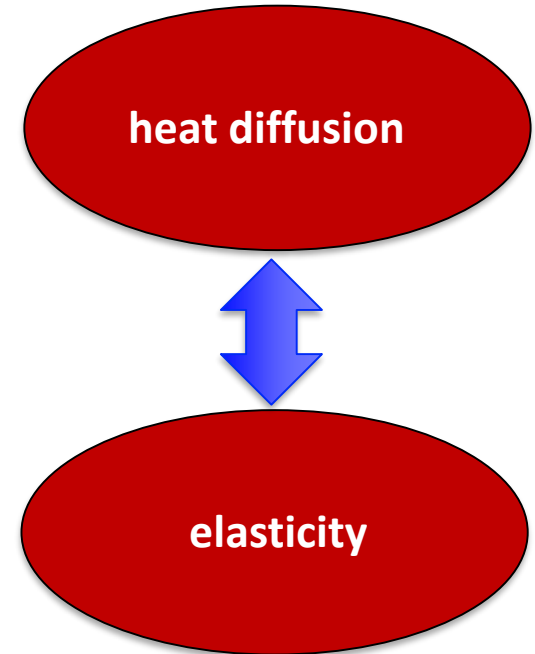
$$\Delta Q = Q_{in} - Q_{out} \qquad \vec{q} = -k\nabla T$$

- ▶ If the internal energy does not depend only on temperature:

$$\rho c \frac{\partial T}{\partial t} + T_0 \alpha p - \nabla \cdot (k \nabla T) = \dot{q}_v$$

- ▶ again under assumptions:

- ▶  $(T_1 - T_0)/T_0 \ll 1$
- ▶ All coefficients are independent of temperature and pressure

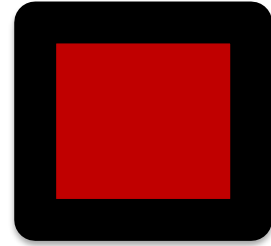




# Thermoelasticity – a simple example

- ▶ How large is the discrepancy?

$$E = \frac{1}{2} \sigma \cdot \epsilon = \frac{1}{2} K \alpha^2 (T_1 - T_0)^2$$



- ▶ Temperature increase of 100 °C results in a discrepancy of

Material	K [MPa]	a [ $10^{-6}/K$ ]	E [J/m <sup>3</sup> ]
Aluminium	70000	23	18.51
Concrete	20000	12	1.44
Water	2200	69	5.24

- ▶ Volumetric compression by  $10^{-6}$  results in elastic energy of 105, 30, 3.3 kJ/m<sup>3</sup> correspondingly





## Closing remarks

- ▶ Take the time to find out the underlying assumptions
- ▶ Don't use models outside their domain of validity
- ▶ Keep in mind where errors can arise and how big they can get
- ▶ Keep an eye on reality
- ▶ No model is perfect, small discrepancies for the sake of convenience can be acceptable